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# Vibration Analysis of Circular Single-layer Graphene Sheet Using Finite Element Method

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## ABSTRACT

Graphene sheets are combined of Honeycombs lattice carbon-carbon bonds which have high natural frequencies, high strength, and high conductivity. Due to important applications of the graphene sheets particularly at higher frequencies, the study of their dynamic behavior is important in this frequency range. From Molecular Dynamics (MD) point of view as the dimensions of graphene sheet incline, the number of atoms increases, and as a result, its modeling becomes more time-consuming. Besides the experimental methods in small dimensions are difficult to conduct and not economical. In this research Finite Element Method (FEM) is used for frequency analysis of graphene sheets in various dimensions in order to study the capability of FEM in simulating the dynamic behavior of graphene sheets at small scales. In this research, the objective function is to find the minimum size of the sheet in which both methods have good convergence. Also, the time-consuming for the simulation is investigated. The time-consuming for analysis in the Finite Element Method is less than other methods, including Molecular Dynamics (MD), Generalized Differential Quadrature (GDQ), etc. Also, The results indicated that for circular single-layer graphene sheets simulation, using Finite Element Method (FEM) is in good agreement with the results obtained from the Molecular Dynamics (MD) simulation, in the radius more than 100 nm. In this research, the ABAQUS has been used for Finite Element Method (FEM) simulation.

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# 1. Introduction

Graphene, as one of the main allotropes of carbon in the single-layer state, is made of honeycombs lattice carbon-carbon bonds. This lattice has a very high natural frequency and also high strength and conductivity and can have higher properties in an electric field. The recent applications of graphene have been in different fields such as large scale measurement devices. sensors, transparent electrodes, solar cells, energy storage devices, composites, and polymer nanocomposites. Regarding the widespread applications of graphene at higher frequencies, the study of its different features particularly its dynamic behavior at this

frequency range is important [1]. Recently, several studies have been conducted to clarify the dynamic behavior of graphene. In some of these investigations, the analytical methods have been used to study the vibrations of graphene sheets while in some others Molecular Dynamics simulations have been applied to derive the dynamic behavior of graphene. Leissa and Narita [2] studied the effect of Poisson's coefficient on the natural frequencies of a simply supported circular plate. Yongqiang and Jian [3] considered the effect of the different boundary conditions and different degrees of freedom on the natural frequencies of graphene. Aghababaei and Reddy [4] studied the

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vibrational properties of a single layer graphene sheet using third-order non-local shear deformation theory. Arash and Wang [5] used both non-local continuum mechanics theory and molecular dynamics simulation to study the free vibrations of single layer and bi-layer graphene sheet and compared the results. Murmu and Pradhan [6] performed non-local elasticity theory to study the vibrational properties of single-layer graphene sheets and determined the effect of elastic circumstances on the first mode frequency of singlelayer graphene sheets. Neek-Amal and Peeters [7] investigated the effect of radial loads on buckling and stiffness properties of circular graphene singlelayer sheet using Molecular Dynamics model. Mahmoudinezhad and Ansari [8] modeled the vibrational properties of circular and square singlelayer graphene sheet using Finite Element Method (FEM). Asemi and Farajpour [9] studied the effects of thermal changes on vibrational behavior of circular graphene sheets. Gong et al. [10] modeled the circular graphene sheets and studied the vibrations of mass sensors. They also compared the natural frequencies of these sheets with the results of the Rayleigh method and investigated the effects of temperature and stress on the natural frequencies. Natsuki et al. [11] obtained the natural frequencies of bilayer graphene sheets. Mohammadi et al. [12] modeled the circular and annular graphene sheets with various boundary conditions by deriving their analytical equations. Mortazavi et al. [13] investigated the thermal conductivity of graphene epoxy nanocomposites using a hybrid method of FEM and molecular dynamics. Ansari et al. [14] used a non-local model in small dimensions to determine the characteristics of Multi-laver graphene sheet vibrational properties in the elastic area for different boundary conditions. In this simulation, FEM was used to investigate the effect of length and modulus of elasticity on the vibrational behavior of graphene laminated sheets. Rouhi and Ansari [15] designed an atomic model of singlelayer graphene sheet using FEM in order to study its vibrational behavior. Based on their model, the natural frequencies of the graphene sheet was obtained with different boundary conditions and various dimensions. This study indicated that the vibrational behavior of graphene sheet could be modeled by using Molecular Dynamics simulation, numerical solution of the governing equations or analytical methods. The implementation of these methods requires the various stages, such as: extracting governing equations, a computational method and analyzing the vibrational signals. Molecular Dynamics (MD) method is commonly

used to simulate the vibrational behavior of nanoscale sheets. Based on the literature [12], the accuracy of MD method is verified. When the size of the sheet is enlarged (for example, a circular sheet with a radius of more than 10 nanometer), the CPU time is usually prolonged, due to the increase in the number of particles. Based on the other researches [16], it is possible to simulate nano-sheets in nanoscale by a Finite Element (FE) method (without considering non-local coefficients in the FE simulation). Although, the main problem is to find the minimum dimension of nano-sheet, which the results of Finite Element method would have good agreement with Molecular Dynamics method. The main objective of this paper is to find the minimum size of the circular sheet in which both methods have good convergence. Furthermore, the solving time is presented in both finite element and molecular dynamics method. For this purpose, At the First, the theory of this problem is explained, then the effect of mesh resizing on the convergence of the solutions for the natural frequency values is checked and the best mesh size is suggested. Finally, the simulation results are verified. The result of FE modeling in various dimensions with those of the Molecular Dynamics modeling and the results obtained by other researchers would be compared. Furthermore, the simulation CPU time is studied.

# 2. Theory and Governing Equations

2.1. Free vibrations of a single-layer circular sheet based on the non-local theory

The graphene single-layer sheet displacement equation based on the non-local method is given by Eq. (1) [12]:

$$D\nabla^{2}(\nabla^{2}w) + K_{w}w - K_{G}\nabla^{2}w - K_{w}(e_{0}a)^{2}\nabla^{2}w + K_{G}(e_{0}a)^{2}\nabla^{2}(\nabla^{2}w) - \rho h(e_{0}a)^{2}\nabla^{2}\left(\frac{\partial^{2}w}{\partial t^{2}}\right) + \rho h\frac{\partial^{2}w}{\partial t^{2}} = 0$$
(1)

In this equation, w is the transverse displacement,  $e_0a$  is the non-local coefficient,  $K_w$  and  $K_G$  are the Winkler modulus, and  $\nabla^2$  is the laplacian operator. It is worth to mention that if  $e_0a = 0$ , Eq. (1) transforms into a classic one. Also, D is the flexural strength calculated according to Eq. (2) [12];

$$D = \frac{Eh^3}{12(1-v^2)}$$
(2)

where E is the modulus of elasticity, h is the sheet thickness and  $\upsilon$  is the Poisson coefficient. The displacement of the sheet can be calculated from Eq. (3) [12]:

$$w(r,\theta,t) = w(r,\theta)e^{i\omega t}$$
(3)

where  $i = \sqrt{-1}$  and  $\omega$  indicates the natural frequency.

By inserting Eq. (2) into Eq. (1) and using Eq. (3), the following would be derived [12]:

$$D\nabla^2(\nabla^2 w) + \gamma^2 \nabla^2 w - \lambda^4 = 0$$
<sup>(4)</sup>

In which  $\gamma^2$  and  $\lambda^4$  are obtained as follows [12],

$$\gamma^{2} = \frac{\left(\frac{\rho h(e_{0}a)^{2}\omega^{2}}{D} - \frac{K_{G}}{D} - \frac{K_{W}(e_{0}a)^{2}}{D}\right)}{\left(1 + \frac{K_{G}(e_{0}a)^{2}}{D}\right)}$$
(5)

$$\lambda^{4} = \frac{\left(\frac{\rho h \omega^{2}}{D} - \frac{K_{w}}{D}\right)}{\left(1 + \frac{K_{G}(e_{0}a)^{2}}{D}\right)}$$
(6)

By solving the eigenvalue problem given by Eq. (4), the values of the dimensionless natural frequencies are obtained as Eq. (7) [12]:

$$\Omega = \sqrt{\frac{\rho h}{D}} \omega R_1^2 \tag{7}$$

## 2.2. Finite Element Model

FEM is one of the numerical solution methods of Partial Differential Equations (PDEs) and also integral equations. In this method, the numerical solution of equations is given by converting the partial differential equations into Ordinary Differential Equations (ODEs). The aim of solving partial differential equations is to achieve a simple and stable equation so that it does not lead to inaccurate and unreasonable results. This method solves the problem by dividing a continuous domain into subdomains called elements. Researchers have invented various types of element especially triangular and quadrilateral elements, some of them are very complex. According to the Kirchhoff law, shear deformation is neglected in thin plates. Using equilibrium relations and assuming that the flexural strength, D, and the force, q, are constant, the differential equation of the fourth order is obtained as [17]:

$$D\left(\frac{\partial^4 w}{\partial x^4} + 2\frac{\partial^4 w}{\partial x^2 \partial y^2} + \frac{\partial^4 w}{\partial y^4}\right) - q = 0$$
(8)

If the mechanical behavior of the material is homogeneous, the flexural stiffness can be obtained from Eq. (2) [17].

In this research, square elements with 4 nodes are used. This element has 12 degrees of freedom in each node; therefore in order to approximate the freemotion of  $\mathbf{w}$ , a polynomial with 12 parameters are implemented (Eq. (9)) [17].

$$w = \alpha_{1} + \alpha_{2}x + \alpha_{3}y + \alpha_{4}x^{2} + \alpha_{5}xy + \alpha_{6}y^{2} + \alpha_{7}x^{3} + \alpha_{8}x^{2}y + \alpha_{9}xy^{2} + \alpha_{10}y^{3} + \alpha_{11}x^{3}y + \alpha_{12}xy^{3}$$
(9)

where  $\alpha$  coefficients are constant and are obtained considering the boundary conditions. Also, considering the deflection, w, in each of four nodes of the element, the shape function is determined as follows (10):

$$N_{i}^{T} = \frac{1}{8} (1 + \xi_{0}) (1 + \eta_{0}) \begin{cases} 2 + \xi_{0} + \eta_{0} - \xi^{2} - \eta^{2} \\ b \eta_{i} (1 - \eta^{2}) \\ -a \xi_{i} (1 - \xi^{2}) \end{cases}$$
(10)

In this Equation, a and b are constant coefficients and depend on the type of element [17]. Also,  $\xi_i$  and  $\eta_i$  are the local coordinates of each element node, and  $\xi$  and  $\eta$  are local coordinate axes and are defined as Eqs. (11) and (12).

$$\xi_0 = \xi \xi_i \tag{11}$$

$$\eta_0 = \eta \eta_i \tag{12}$$

Considering the above equations, the stiffness matrix of each element can be calculated as Eq. (13) [17]:

$$K^{e} = \int B^{T} DB dx dy$$
<sup>(13)</sup>

B Matrix is defined as follows:

$$B = [B_i] = [B_1, B_2, B_3, B_4] = (L\nabla)N$$
(14)

In this equation, L is the Lagrangian and N is a shape function of the element [17].

# 3. Simulation

At first, according to the mechanical properties of graphene, the sheet is modeled by ABAQUS. In the present work, the vibrational properties of the circular graphene sheet are examined in different dimensions. This result is compared with those of other studies. Another simulation is implemented using molecular dynamics (MD) model. Taking into account intermolecular forces, MD model has a high degree of accuracy in calculating the behavior of nano-scale sheets. In this research, MD and FE methods are compared. As a result, the best simulation method is represented, based on accuracy and CPU time. In order to model in terms of Finite Element (FE), square element with 4 nodes has been selected in this research. In this type of element, each node has 3 degrees of freedom. Therefore, the square element has 12 degrees of freedom. It is worth to mention that the strain matrix is not fixed in linear square elements; therefore this type of element, gives more accurate results for the stress and strain matrices. Also, the governing equations are simpler due to their regular and symmetrical geometry. Therefore, the CPU time of analysis decreases.

#### 3.1. Mechanical Properties of the Circular Sheet

In order to calculate the natural frequencies and mode shapes of the circular graphene sheet, ABAQUS is implemented. The properties of this sheet are given in Table 1. In this table,  $\rho$  refers to mass density, E is elasticity modulus, h is the thickness of graphene sheet and v is Poisson's ratio.

#### 3.2. Mesh Convergence

The circular sheet is simulated using ABAQUS, according to the properties listed in Table 1. In order to achieve the desired accuracy, the optimum number of elements is selected. After choosing the best type of element, the simulation is done. The meshed graphene sheet is shown in Fig. 1. This figure is a sample. The optimum number of elements should be selected.

 $\label{eq:table1} \textbf{Table 1.} Mechanical properties of the single-layer graphene$ 

	sheet [12].		
$\rho(Kg/m^3)$	E(TPa)	h(nm)	υ
2300	1.06	0.34	0.33



Fig. 1. Graphene Elemental Model.

In order to ensure that the results of the Finite Element (FE) Model are accurate, the convergence analysis is implemented, and the effect of change in number of elements on natural frequencies in the simulated model was checked. Obviously, as the number of elements increases, the size of the elements becomes smaller. This study has been carried out for various sheet size. For example; the variations diagram of the third and sixth natural frequencies of the circular sheet with a radius of 20 nm have been presented in Figs. 2 and 3. The most suitable mesh size of 8400 elements has been suggested for the computation of natural frequencies by taking into account the lowest error value.



Fig. 2. Study the Effect of number of elements on the third natural frequency.



Fig. 3. Study the Effect of number of elements on the sixth natural frequency.

# 4. Results and Discussion

## 4.1. Results of Finite Element Model

In this section, after achieving desire accuracy, the results of a circular graphene sheet analysis, with the properties given in Table 1, are presented. The natural frequencies and mode shapes of this sheet are shown.

For validation, the comparison of the results of this work with those of another study is indicated in Table 2. The value of the non-local coefficient is set according to the desired frequencies. This comparison is executed for sheets with the radius of 6, 7, 8, 10, 15 and 20 nm.

 Table 2. Comparison of natural frequencies (GHz) of the circular sheet with simply support boundary conditions.

radius	M	Mohammadi et		Finite	
	Number	al. [12]		Element	error
(nm)	01 modoc		English	(present	(%)
	moues	$e_0a$	Frequency	study)	
	1	0	49.232	49.0878	0.292
	2	0.5	136.2383	136.828	0.433
	3	0.5	136.2383	136.828	0.433
6	4	0.5	247.7065	249.741	0.821
	5	0	255.5455	251.037	1.764
	6	0	296.5175	291.667	1.635
	1	0	36.1705	36.1107	0.165
	2	0.5	100.0935	100.863	0.768
	3	0.5	100.0935	100.863	0.768
7	4	0.5	181.9884	184.524	1.393
	5	0	187.7477	185.655	1.114
	6	0	217.8496	215.590	1.037
	1	0	27.693	27.7846	0.33
	2	0	77.9978	78.0700	0.092
_	3	0	77.9978	78.3064	0.395
8	4	0	143.7443	143.440	0.211
	5	0	143.7443	146.056	1.608
	6	0	166.7911	169.268	1.485
	1	0	17.7235	17.7308	0.041
	2	0	49.9186	49.7199	0.398
	3	0	49.9186	49.7792	0.279
10	4	0	91.9964	91.3311	0.723
	5	0	91.9964	92.1862	0.206
	6	0	106.7463	107.027	0.262
15	1	0	7.791	7.85775	0.856
	2	0	21.9436	21.9545	0.049
	3	0	21.9436	21.9545	0.049
	4	0	40.4404	40.3246	0.286
	5	0	40.4404	40.3246	0.286
	6	0	46.9243	46.8451	0.168
20	1	0	4.3825	4.4217	0.894
	2	0	12.3433	12.3658	0.182
	3	0	12.3433	12.3660	0.183
	4	0	22.7477	22.7334	0.062
	5	0	22.7477	22.7447	0.021
	6	0	26.3949	26.4083	0.050

Table 3. Comparison of natural frequencies (GHz) of the simply supported sheet with radius 20 (nm) with other references.

000000				
Numbe r of	Reference	Referenc	Referenc	Finite Element
madag	[12]	e [2]	[18] e	(presen
moues				t study)
1	4.3825	4.38308	4.38308	4.4217
2	12.3433	12.3432	12.3432	12.3658
3	12.3433	12.3432	12.3432	12.3660
4	22.7477	22.7476	22.7493	22.7334
5	22.7477	22.7476	22.7493	22.7427
6	26.3949	26.3951	26.3942	26.4083



Fig. 4. Mode shape of a circular sheet with the radius of 20 nm from FEM model: a) Mode 1, b) Mode 2, c) Mode3, d) Mode 4, e) Mode 5 and f) Mode 6.

In order to ensure the accuracy of the results, the natural frequencies of the circular sheet with the radius of 20 nm are validated against several previous works which are summarized in Table 3.

In addition to the natural frequencies, the mode shapes of the single-layer graphene sheet are also obtained, which are shown in Fig. 4, for a sheet with the radius of 20 nm.

#### 4.2. Results of Molecular Dynamics Model

The aim of this section is to find the lowest radius of the circular single-layer graphene sheet, which corresponds to the natural frequency calculated by FEM compared to the natural frequency obtained from the MD method. Molecular Dynamics (MD) has been suggested in studies that are costly and time-consuming. This method has often been applied in very small dimensions by researchers. According to the physical laws, simulation using Molecular Dynamics (MD) has been conducted by determining the type, position, properties of atoms and the force between them. In the following, the Molecular Dynamics (MD) model of the circular graphene sheet is also carried out, and its results are compared with those of FEM. A circular graphene sheet that is modeled by Molecular Dynamics method is shown in Fig. 5.

For simulating the graphene sheet in the Molecular Dynamics (MD) model, Carbon-Carbon bonds are simulated by Tersoff potential model. The simulation is executed in room temperature. As the initial condition, a pulse is applied to graphene sheet by step-time of 0.001 ps. In this method, the particle displacement is considered as an output [19]. In the frequency decomposition method that is presented by Brincker et al. [20], the natural frequencies and mode shapes are extracted after estimation of spectral density matrix and applying the decomposition method for the singular values on it. For this purpose, after ABAQUS simulation, the first natural frequency of the sheet with different radius is compared with MD Model and the results are summarized in Table 4. Also, the CPU time of simulation using both methods is presented in this table. The configuration of the system that is utilized for the simulation is Intel® Xeon® Processor X5675 (12M Cache, 3.06 GHz, 6.40 GT/s Intel® QPI) FC-LGA10) with Random Access Memory as 64 GB.



**Fig. 5.** A circular Graphene sheet with the radius of 8 nm that is modeled by Molecular Dynamic Method in LAMMPS software.

**Table 4.** Comparison of the first mode frequency (GHz) of the sheet with simply supported boundary condition in two Finite Element (FE) Models and Molecular Dynamics (MD) Models.

FEM		MD		
Frequency (GHz)	Time- consuming (min)	Frequency (GHz)	Time- consuming (min)	ref. [14]
27.7846	0.11	39.0625	372	27.693
4.4263	0.23	13.4277	1254	4.3825
1.9674	0.43	8.5449	4856	1.8234
1.1062	0.64	7.3242	6249	0.9576
0.49161	0.82	4.8828	7203	0.4365
	Frequency (GHz) 27.7846 4.4263 1.9674 1.1062 0.49161	FEM           Frequency (GHz)         Time- consuming (min)           27.7846         0.11           4.4263         0.23           1.9674         0.43           1.1062         0.64           0.49161         0.82	FEM         M           Frequency (GHz)         Time- consuming (min)         Frequency (GHz)           27.7846         0.11         39.0625           4.4263         0.23         13.4277           1.9674         0.43         8.5449           1.1062         0.64         7.3242           0.49161         0.82         4.8828	FEM         MD           Frequency (GHz)         Time- consuming (min)         Frequency (GHz)         Time- consuming (min)           27.7846         0.11         39.0625         372           4.4263         0.23         13.4277         1254           1.9674         0.43         8.5449         4856           1.1062         0.64         7.3242         6249           0.49161         0.82         4.8828         7203

As a result of the comparison, the first natural frequency of FE and MD models versus radius of the graphene sheet is plotted in Fig. 6. According to this figure, the predicted radius that results of FE and MD models are matched is 100 nm. In low radius, the non-local effects are high. The Finite Element Model does not consider these effects. Therefore, the difference between MD and FE models in low radius is high. This difference in the radius greater than 100 nm is reduced to a desirable level. Also, according to table 4, the CPU time for MD simulation is more than FE simulation. Simulation time for graphene sheets with radius of higher than 20 nm, take long more than one day, although this time for FE simulation is less than one minute. According to table 4 in comparison with table 2, the non-local coefficient by increasing the radius of the sheet tends to zero. In other words, by increasing the sheet size, the effects of intermolecular forces decrease as well as the difference between natural frequencies of MD and FE simulation decreases.

According to Table 2 and 3, the second and third modes, as well as the fourth and fifth modes, are doubled modes. As shown in Fig. 6, whatever the radius of the sheet increased, the FE and MD results became closer together. It's due to a reduction in the non-local effects in Molecular Dynamics (MD) model, Also the model is approached to the classic model. Therefore, due to the time-consuming simulation by Molecular Dynamics (MD) model, results larger than 60 nm radius is predicted by extrapolation. Consequently, in large dimensions, graphene single-layer circular sheet, due to a large number of atoms, simulation by Molecular Dynamics (MD) is very time consuming, and ABAQUS can be used for this modeling.

## 5. Conclusions

In this article, the dynamic behavior of circular graphene sheets has been investigated using FE and MD methods. In order to guarantee the accuracy of the results of the FE method, the natural frequencies of the nano-sheet are verified by those of other studies [12]. Also, the effect of sheet size on the accuracy of simulation by Finite Element is studied. This research aimed to find the minimum size of nano-sheet, which the results of finite element method would have good agreement with molecular dynamics method. The main objective of this paper is to find the minimum size of the circular sheet in which both methods have good convergence. Furthermore, the solving time is presented in both finite element and molecular dynamics method. Generally, the results of this study can be summarized as follows:

- 1. The results showed that the Finite Element results in the radius of 8 nm are a better adapted to the classic sheet model.
- 2. The CPU time of simulation in the molecular dynamics method in large scales is very high, due to the increase in the number of atoms.
- 3. The results of the frequency analysis of the ABAQUS with a radius of 100 nm and greater are close to the Molecular Dynamics results.

# References

 Singh V, Joung D, Zhai L, Das S, Khondaker SI, Seal S. Graphene based materials: past, present and future. *Progress in materials science* 2011; 56(8): 1178-271.

- [2] Leissa AW, Narita Y. Natural frequencies of simply supported circular plates. *Journal of Sound and Vibration* 1980; 70(2): 221-9.
- [3] Yongqiang L, Jian L. Free vibration analysis of circular and annular sectorial thin plates using curve strip Fourier p-element. *Journal of Sound and Vibration* 2007; 305(3): 457-66.
- [4] Aghababaei R, Reddy JN. Non-local third-order shear deformation plate theory with application to bending and vibration of plates. *Journal of Sound and Vibration* 2009; 326(1): 277-89.
- [5] Arash B, Wang Q. Vibration of Single- and Double-Layered Graphene Sheets. *Journal of Nanotechnology in Engineering and Medicine* 2011; 2(1): 011012--7.



Fig. 6. Compare the frequencies of the two methods and predict their matching: (a) Mode 1, (b) Mode 2, (c) Mode 3, (d) Mode 4, (e) Mode 5 and (f) Mode 6.

- [6] Murmu T, Pradhan SC. Vibration analysis of nano-single-layered graphene sheets embedded in elastic medium based on nonlocal elasticity theory. *Journal of Applied Physics* 2009; 105(6): 064319.
- [7] Neek-Amal M, Peeters FM. Buckled circular monolayer graphene: a graphene nano-bowl. *Journal of Physics: Condensed Matter* 2011; 23(4): 045002.
- [8] Mahmoudinezhad E, Ansari R. Vibration analysis of circular and square single-layered graphene sheets: An accurate spring mass model. *Physica E: Low-dimensional Systems and Nanostructures* 2013; 47: 12-6.
- [9] Asemi SR, Farajpour A. Decoupling the nonlocal elasticity equations for thermomechanical vibration of circular graphene sheets including surface effects. *Physica E: Lowdimensional Systems and Nanostructures* 2014; 60: 80-90.
- [10] Gong X, Jiang S, Wang X, Liu S, Wang S. Vibration analysis of nanomechanical mass sensor based on circular graphene sheets2014.
- [11] Natsuki T, Shi J-X, Ni Q-Q. Vibration analysis of circular double-layered graphene sheets. *Journal of Applied Physics* 2012; 111(4): 044310.
- [12] Mohammadi M, Ghayour M, Farajpour A. Free transverse vibration analysis of circular and annular graphene sheets with various boundary conditions using the non-local continuum plate model. *Composites Part B: Engineering* 2013; 45(1): 32-42.

- [13] Mortazavi B, Benzerara O, Meyer H, Bardon J, Ahzi S. Combined molecular dynamics-finite element multiscale modeling of thermal conduction in graphene epoxy nanocomposites. *Carbon* 2013; 60: 356-65.
- [14] Ansari R, Rajabiehfard R, Arash B. Non-local finite element model for vibrations of embedded multi-layered graphene sheets. *Computational Materials Science* 2010; 49(4): 831-8.
- [15] Rouhi S, Ansari R. Atomistic finite element model for axial buckling and vibration analysis of single-layered graphene sheets. *Physica E: Low-dimensional Systems and Nanostructures* 2012; 44(4): 764-72.
- [16] Tserpes K, Papanikos P. Finite element modeling of single-walled carbon nanotubes. *Composites Part B: Engineering* 2005; 36(5): 468-77.
- [17] Mirzaei M, Rouzegar J. Crack simulation in shells and plates using Extended Finite Element Method. *Modares Civil Engineering* 1389; 10(4): 1-12.
- [18] Zhou ZH, Wong KW, Xu XS, Leung AYT. Natural vibration of circular and annular thin plates by Hamiltonian approach. *Journal of Sound and Vibration* 2011; 330(5): 1005-17.
- [19] Rapaport DC. The Art of Molecular Dynamics Simulation: Cambridge University Press; 1995.
- [20] Rune B, Lingmi Z, Palle A. Modal identification of output-only systems using frequency domain decomposition. *Smart Materials and Structures* 2001; 10(3): 441.